

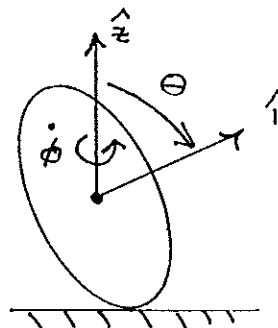
DUE: THURSDAY DEC 8, 1988 ; MAXIMUM RECORDED SCORE = 50 POINTS

① SPINNING COIN REVISITED

IF YOU SPIN A COIN ON A TABLE, YOU CAN MAKE IT SPIN ABOUT A VERTICAL DIAMETER AS THE AXIS, WITH THE C.M. AT REST. IF THE ANGULAR VELOCITY BECOMES 'TOO LOW', THE COIN FALLS OVER INTO THE MOTION OF PROB. ⑥, SET 9.

CONSIDER A DISC OF MASS M , RADIUS a WHICH SPINS WITHOUT FRICTION ON A HORIZONTAL SURFACE (\Rightarrow C.M. CAN ONLY MOVE VERTICALLY). USE EULER'S ANGLES AND LAGRANGE'S METHOD TO FIND THE STEADY PRECESSION RATE $\dot{\phi}$ AS A FUNCTION OF THE ANGLE Θ OF THE SYMMETRY AXIS TO THE VERTICAL, AND OF $\omega_1 =$ ANGULAR VELOCITY ABOUT THE SYMMETRY AXIS

$$\text{SHOW } \dot{\phi}_{\text{STEADY}} = \frac{\omega_1 \pm \sqrt{\omega_1^2 + \frac{4g \cos^2 \Theta}{a \sin \Theta}}}{\cos \Theta}$$



AS $\Theta \rightarrow \pi/2$ (COIN SPINNING ON EDGE) THE ABOVE SOLUTION BREAKS DOWN! SHOW THAT WHEN $\Theta = \pi/2$ THE TWO POSSIBLE

MOTIONS ARE $\left\{ \begin{array}{l} \omega_1 \text{ ARBITRARY, } \dot{\phi} = 0 \Leftrightarrow \text{'ROLLING \& SLIPPING'} \\ \omega_1 = 0, \dot{\phi} \text{ ARBITRARY} \Leftrightarrow \text{'SPINNING ON EDGE'} \end{array} \right.$

WE ARE INTERESTED IN THE 2ND CASE.

USE THE SMALL ANGLE APPROXIMATION ($\Theta = \pi/2 - \epsilon$)

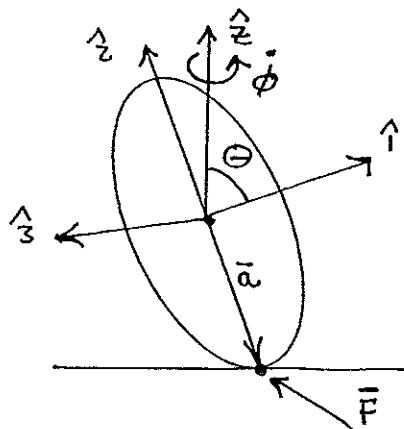
TO SHOW THE 2ND MOTION IS STABLE ONLY IF

$$\dot{\phi} > 2 \sqrt{\frac{g}{a}}$$

OTHER WISE THE COIN FALLS OVER ...

② ROLLING DISC REVISITED

WE CONSIDER ARBITRARY MOTION OF A DISC ROLLING WITHOUT SLIPPING ON A HORIZONTAL PLANE. EXPERTS WILL NOTE THAT THIS IS A NON-HOLONOMIC PROBLEM WITH 5 COORDINATES AND 2 CONSTRAINTS. BUT TRY IT WITHOUT RESORT TO LAGRANGE MULTIPLIERS!



THE DISC HAS MASS m , RADIUS a

CHOOSE A COORDINATE SYSTEM SIMILAR TO, BUT NOT QUITE THE SAME, AS EULER'S.

$\hat{1}$ ALONG THE SYMMETRY AXIS

$\hat{2}$ VERTICAL

$\hat{2}$ IN THE PLANE OF THE DISC, AND IN $\hat{1}$ - $\hat{2}$ PLANE

$\hat{3} = \hat{1} \times \hat{2}$ IS HORIZONTAL, $\dot{\phi}$ IN PLANE OF DISC.

THIS IS A PRINCIPAL AXIS COORD. SYSTEM \Rightarrow INERTIA TENSOR DIAGONAL!
BUT IT IS NOT QUITE THE 'BODY FRAME'!

$\Theta =$ ANGLE BETWEEN $\hat{1} \notin \hat{z}$

$\dot{\phi} =$ ROTATION RATE OF THE DISC ABOUT THE VERTICAL.

$\vec{F} =$ UNKNOWN FORCE ON THE DISC AT THE POINT OF CONTACT.

THE ELEMENTARY EQUATIONS OF MOTION ARE, OF COURSE,

$$\vec{F}_{\text{TOTAL}} = m \frac{d\vec{v}}{dt} \quad \text{WHERE } \vec{v} = \vec{v}_{\text{OF CM}} \quad ; \quad \vec{N}_{\text{C.M.}} = \frac{d\vec{L}_{\text{C.M.}}}{dt}$$

THE CONSTRAINT OF ROLLING WITHOUT SLIPPING IS EASILY WRITTEN DOWN IN TERMS OF VELOCITIES:

$$\vec{v}_{\text{OF CONTACT}} = 0 = \vec{v} + \vec{\omega} \times \vec{a} \quad (\text{CHASLES' THEOREM})$$

WHERE $\vec{a} =$ VECTOR FROM C.M. TO POINT OF CONTACT

$$\vec{\omega} = \text{TOTAL ANGULAR VELOCITY} = \vec{\omega}_a + \dot{\psi} \hat{1}$$

$\vec{\omega}_a =$ ANGULAR VELOCITY OF THE AXES

$\dot{\psi} =$ ANGULAR VELOCITY OF THE DISC RELATIVE TO THE $\hat{1}, \hat{2}, \hat{3}$ AXES

DEFINE $\omega_1 \equiv \dot{\psi} + \omega_{a1}$ TO AVOID CARRY BOTH SYMBOLS THRU THE ENTIRE PROBLEM.

ELIMINATE \bar{F} AND \bar{v} TO GET THE EQUATION OF MOTION

$$\frac{m a^2}{4} \frac{d}{dt} (2\omega_1 \hat{1} + \dot{\phi} \sin \theta \hat{2} - \dot{\theta} \hat{3}) = m g a \cos \theta \hat{3} - m a^2 \hat{2} \times \frac{d}{dt} (\dot{\theta} \hat{1} + \omega_1 \hat{3})$$

FIRST CONSIDER STEADY MOTION $\Rightarrow \omega_1, \dot{\phi} = \text{CONSTANT}$
 $\dot{\theta} = 0$

NOTE THAT FOR THE AXES, $\frac{d\hat{c}}{dt} = \bar{\omega}_a \times \hat{c}$ TO SHOW THAT

$$\dot{\phi}^2 \sin^2 \theta \cos \theta - 6 \omega_1 \dot{\phi} \sin \theta - \frac{4g}{a} \cos \theta = 0$$

[TO RELATE THIS TO PROB ⑤, SET 9, NOTE THAT $a\omega_1 = -b\dot{\phi}$ FOR ROLLING IN A CIRCLE OF RADIUS b , AND THAT θ IS DEFINED DIFFERENTLY.]

AT $\theta = \pi/2$ THE DISC IS ON EDGE.

WE ARE INTERESTED IN THE ROLLING SOLUTION; $\dot{\theta} = 0$, BUT ω_1 ARBITRARY.

IS THIS MOTION STABLE?

THAT IS, CONSIDER $\theta = \pi/2 - \epsilon$, $\dot{\phi}$ SMALL AND ω_1 ARBITRARY.

PLUG INTO THE EQUATION OF MOTION AND IGNORE 2ND ORDER TERM: $\epsilon^2, \dot{\epsilon}\dot{\phi}$ ETC.

REMEMBER $\frac{d\hat{c}}{dt} = \bar{\omega}_a \times \hat{c}$

SHOW THAT THE $\hat{1}$ TERMS $\Rightarrow \dot{\omega}_1 = 0, \omega_1 = \text{CONST}$

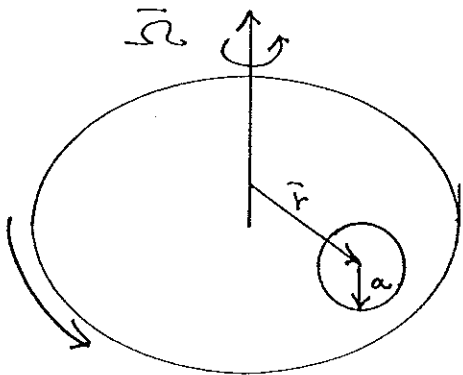
$\hat{2}$ TERMS $\Rightarrow \dot{\phi} = -2\omega_1 \epsilon$

$\hat{3}$ TERMS $\Rightarrow \ddot{\epsilon} = -\frac{4}{5} (3\omega_1^2 - \frac{g}{a}) \epsilon$

SO THE ROLLING IS STABLE IF $\omega_1 > \sqrt{g/3a}$

IF $\omega_1 < \sqrt{g/3a}$ THE DISC FALLS OVER AND GOES INTO THE MOTION OF PROB ⑥, SET 9. TRY IT WITH A COIN, A HULA HOOP...

③ MARBLE ROLLING ON A TURNTABLE (SO THAT'S HOW YOU LOST YOUR MARBLES)



A MARBLE OF MASS m , RADIUS a (A UNIFORM SPHERE) ROLLS WITHOUT SLIPPING ON A HORIZONTAL TURNTABLE. THE TURNTABLE ROTATES WITH CONSTANT ANGULAR VELOCITY $\vec{\Omega}$ ABOUT THE VERTICAL.

USE THE APPROACH OF PROB ② TO ANALYZE THE MOTION BY 'ELEMENTARY' METHODS IN THE LAB FRAME.

a) WHAT IS THE CONSTRAINT RELATION BETWEEN $\vec{\Omega}$, \vec{r} , \vec{v} , \vec{a} AND $\vec{\omega}$?

- \vec{r} = C.M. POSITION OF MARBLE
- \vec{v} = C.M. VELOCITY OF MARBLE
- \vec{a} = RADIUS VECTOR FROM THE C.M. TO POINT OF CONTACT
- $\vec{\omega}$ = TOTAL ANGULAR VELOCITY,

b) IT SEEMS REASONABLE THAT A POSSIBLE MOTION IS JUST MOTION IN A CIRCLE OF RADIUS ρ ABOUT THE TURNTABLE AXIS. USE 'ELEMENTARY METHODS' TO SHOW THAT

$$\omega = \frac{\Omega}{a} \left(\frac{ma^2}{I+ma^2} \right) \rho \quad \left[\text{WE NEGLECT A POSSIBLE SPIN OF THE MARBLE ABOUT THE VERTICAL, WHICH IS INDEPENDENT OF } \Omega \dots \right]$$

WHERE I = MOMENT OF INERTIA OF THE MARBLE.

IN WHAT DIRECTION IS $\vec{\omega}$? YOU SHOULD ALSO FIND $\dot{\phi} = \Omega \frac{I}{I+ma^2}$ FOR C.M. MOTION ABOUT THE VERTICAL

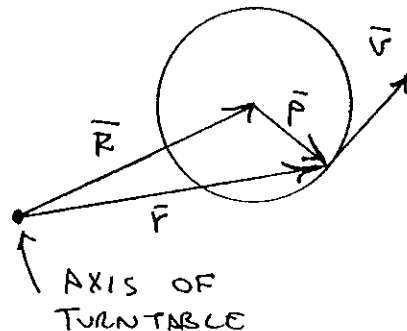
c) CONSIDER ARBITRARY MOTION. NOTE THAT BY DIFFERENTIATING THE CONSTRAINT RELATION, YOU CAN ELIMINATE $d\vec{\omega}/dt$ FROM THE EQUATIONS OF MOTION LEADING TO

$$\frac{I+ma^2}{I} \cdot \frac{d\vec{v}}{dt} = \frac{d}{dt} (\vec{\Omega} \times \vec{r})$$

WITH $\vec{\Omega} = \text{CONST}$, SHOW THAT THIS LEADS TO

$$\vec{v} = \frac{I}{I+ma^2} \vec{\Omega} \times \vec{p} \quad \text{WHERE } \vec{r} = \vec{R} + \vec{p}$$

$$\text{AND } \vec{R} = \vec{v}_0 + \frac{I+ma^2}{I} \frac{\vec{\Omega} \times \vec{v}_0}{\Omega^2}$$

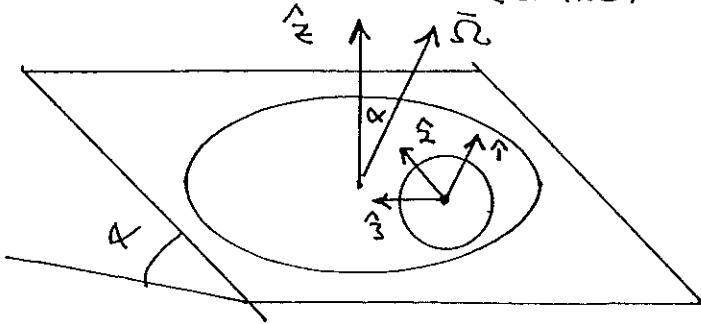


THIS IS CIRCULAR MOTION IN A CIRCLE OF RADIUS ρ ABOUT THE POINT AT \vec{R} .

$\vec{v}_0, \vec{r}_0 =$ INITIAL VELOCITY AND POSITION OF THE MARBLE.

FINALLY, SHOW
$$\vec{\omega} = \frac{\Omega}{a} \frac{I \vec{R} + m a^2 \vec{v}}{I + m a^2}$$

d) SUPPOSE THE PLANE OF THE TURN TABLE MAKES ANGLE α TO THE HORIZONTAL.



TAKE INTO ACCOUNT THE POSSIBILITY THAT THE FORCE \vec{F} OF THE TURN TABLE ON THE MARBLE INCLUDES A NORMAL FORCE $\neq mg$.

CHOOSE AXES $\uparrow \perp$ TO TURN TABLE
 $\hat{2}$ POINTING UP SLOPE
 $\hat{3}$ HORIZONTAL.

FIND THE SLIGHTLY MODIFIED EQUATION OF MOTION FOR

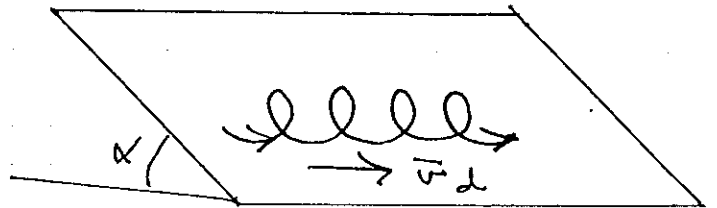
$\frac{d\vec{v}}{dt}$ AS IN PART c)

SHOW THAT A SOLUTION IS $\vec{v} = \vec{v}_c + \vec{v}_d$

WHERE $\vec{v}_c =$ THAT FOUND IN PART c).

$$\vec{v}_d = - \frac{m g a^2 \sin \alpha}{I \Omega} \hat{3} = \text{CONSTANT}$$

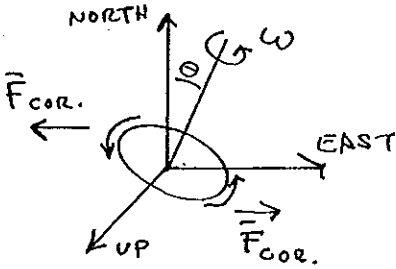
HENCE THE MOTION LOOKS LIKE



WHY NOT TRY IT!

④ GYROCOMPASS

A GYROCOMPASS IS A SPINNING FLYWHEEL WHOSE AXIS OF ROTATION IS CONSTRAINED TO MOVE IN A HORIZONTAL PLANE AT THE SURFACE OF THE EARTH. (NO HORIZONTAL FORCES ARE APPLIED TO THE AXIS).



IF WE ANALYZE THE MOTION IN A FRAME FIXED ON THE EARTH'S SURFACE, WE MUST TAKE THE CORIOLIS FORCE INTO ACCOUNT. WHEN THE AXIS MAKES ANGLE θ TO THE NORTH AS SHOWN, THE LEFT SIDE OF THE FLYWHEEL IS MOVING UP \Rightarrow CORIOLIS FORCE TO THE WEST. THE RIGHT SIDE OF THE FLYWHEEL IS MOVING DOWN \Rightarrow FORCE TO THE EAST.

TOP VIEW

THE WEST. THE RIGHT SIDE OF THE FLYWHEEL IS MOVING DOWN \Rightarrow FORCE TO THE EAST.

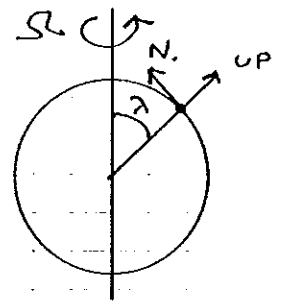
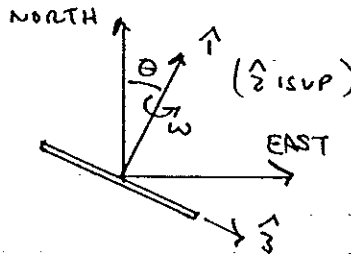
\therefore THERE IS A NET TORQUE ABOUT THE VERTICAL WHICH TENDS TO RESTORE θ TO 0° . I.E., THE GYRO LIKES TO POINT NORTH!

a) ASSUME THE FLYWHEEL IS A HOOP OF RADIUS a . CALCULATE THE TOTAL CORIOLIS TORQUE TO SHOW

$$N_1 = 0$$

$$N_2 = M a^2 \omega \Omega \sin \lambda \sin \theta$$

$$N_3 = M a^2 \omega \Omega \cos \lambda$$



WHERE $\hat{1}, \hat{2}, \hat{3}$ ARE PRINCIPAL AXES AS SHOWN. (THESE ARE NOT BODY AXES, IN THAT $\hat{2}$ IS ALWAYS UP)
 Ω = ROTATION OF EARTH
 λ = CO-LATITUDE OF THE GYROCOMPASS.

SHOW THAT THIS TORQUE CAUSES THE GYRO TO OSCILLATE ABOUT $\theta = 0^\circ$ WITH ANGULAR FREQUENCY $\sqrt{2 \omega \Omega \sin \lambda}$ (FOR SMALL θ_{MAX}).

b) ANALYZE THE GYROCOMPASS FROM THE POINT OF VIEW OF AN INERTIAL OBSERVER.

SUCH AN OBSERVER WOULD SAY $\frac{d\vec{L}}{dt} = \vec{N}$, WHERE TORQUE \vec{N} IS

ONLY DUE TO THE CONSTRAINT FORCES ON THE AXLE, WHICH KEEP IT HORIZONTAL. THESE DON'T MAKE THE GYRO POINT NORTH.

BUT $\vec{L} = \vec{I} \cdot \vec{\omega}_{TOT}$ AND $\vec{\omega}_{TOT}$ HAS 3 PIECES.

- ROTATION ABOUT THE GYRO AXLE BY AMOUNT $\vec{\omega}$
- ROTATION $\dot{\theta}$ IN THE PLANE OF THE SURFACE OF THE EARTH.
- ROTATION OF THE EARTH ABOUT ITS AXIS, $\vec{\Omega}$

EVALUATE $\vec{\omega}_{TOT}$ IN THE INERTIAL FRAME BY GIVING ITS COMPONENTS ALONG THE DIRECTIONS OF THE PRINCIPAL AXES DEFINED ABOVE. ALSO EVALUATE \vec{L} FOR A FLYWHEEL WHICH IS A UNIFORM DISC OF MASS M , RADIUS a .

WITH RESPECT TO THE INERTIAL OBSERVER, THE PRINCIPAL AXES ARE ROTATING WITH ANGULAR VELOCITY $\dot{\Theta} + \vec{\Omega}$.

EVALUATE $\vec{N} = \frac{d\vec{L}}{dt}$.

SHOW THAT THE COMPONENTS OF THIS EQUATION FOR WHICH $N_i = 0$ LEAD TO $\dot{\omega} = 0$ AND $\ddot{\Theta} + 2\omega\Omega(\sin\lambda)\Theta = 0$ FOR $\Theta \ll \Omega$ SMALL.

HENCE THE GYRO OSCILLATES ABOUT $\Theta = 0$ (NORTH) WITH ANGULAR FREQUENCY $\sqrt{2\omega\Omega \sin\lambda}$ AS IN PART a).

IN PRACTICE A MOTOR IS REQUIRED TO KEEP ω CONSTANT. DAMPING IN THE GYRO KILLS THE OSCILLATIONS, LEAVING THE AXIS POINTING NORTH.

OF COURSE, IF THE CONSTRAINING PLANE OF THE GYRO TIPS, THE GYRO WON'T POINT NORTH ANYMORE. A REAL GYRO COMPASS MUST HAVE A MECHANISM TO FIND THE HORIZONTAL PLANE EVEN WHEN ITS SUPPORTS ARE TIPPING — AS ON A SHIP OR AIRPLANE. THIS REQUIRES A PLUMB BOB — AND A MECHANISM TO DEFEAT THE INFLUENCE OF A POSSIBLY OSCILLATING POINT OF SUPPORT....

WELL, IT'S A BIT COMPLICATED.

PH 205 SET 10

⑤ CONSIDER AN ASYMMETRIC TOP WHOSE PRINCIPAL MOMENTS OF INERTIA ARE $I_1 > I_2 > I_3$. WE HAVE CLAIMED THAT ROTATIONS WITH $\bar{\omega}$ NEAR \hat{z} ARE 'UNSTABLE'.

EXAMINE THE SPECIAL CASE WHEN $T = \frac{L^2}{2I_2}$ (= KINETIC ENERGY)

USE THE EXPRESSIONS FOR T AND L^2 TO SHOW

$$\omega_1^2 = \frac{I_2 - I_3}{I_1 - I_3} \frac{L^2 - I_2^2 \omega_2^2}{I_1 I_2} \quad \omega_3^2 = \frac{I_1 - I_2}{I_1 - I_3} \frac{L^2 - I_2^2 \omega_2^2}{I_2 I_3}$$

AND THAT EULER'S EQUATIONS LEAD TO

$$\omega_2 = \omega_{2 \text{ MAX}} \tanh [K \omega_{2 \text{ MAX}} (t + t_0)] \quad \text{WHERE } \omega_{2 \text{ MAX}} = L/I_2$$

$$\text{AND } K = \sqrt{\frac{(I_1 - I_2)(I_2 - I_3)}{I_1 I_3}}$$

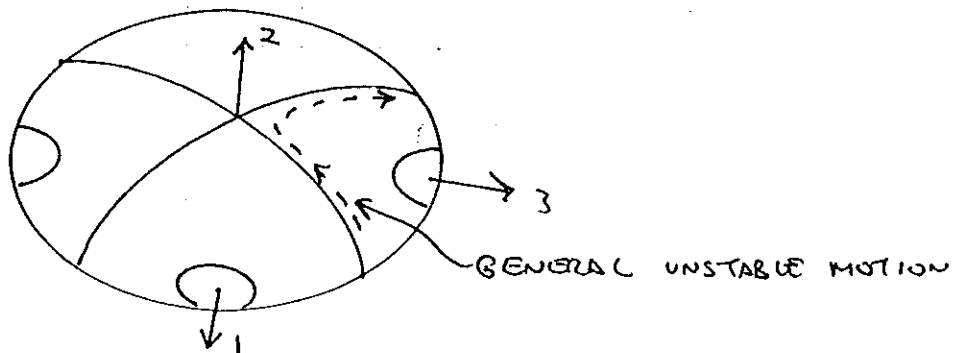
$$\text{AND } \omega_1 = \omega_{1 \text{ MAX}} \sqrt{\frac{I_1 (I_2 - I_3)}{I_2 (I_1 - I_3)}} \operatorname{sech} [K \omega_{2 \text{ MAX}} (t + t_0)]$$

$$\omega_3 = \omega_{3 \text{ MAX}} \sqrt{\frac{I_3 (I_1 - I_2)}{I_2 (I_1 - I_3)}} \operatorname{sech} [K \omega_{2 \text{ MAX}} (t + t_0)]$$

HENCE AS $t \rightarrow \infty$, $\omega_1, \omega_3 \rightarrow 0$, $\omega_2 \rightarrow \omega_{2 \text{ MAX}}$

AND THE ROTATION IS ABOUT AXIS 2.

THUS FOR THIS SPECIAL CASE OF MOTION ALONG THE 'SEPARATING POLYNODES', A KIND OF STABILITY OCCURS.



IN PRACTICE THIS IS HARD TO ACHIEVE, SINCE IF $T \neq \frac{L^2}{2I_2}$

$\bar{\omega}$ WILL MOVE TOWARDS \hat{z} & THEN 'BOUNCE' AWAY, MOVING TOWARDS $-\hat{z}$, ROUGHLY ALONG THE OTHER SEPARATING POLYNODE ...